

REMARKS

Applicant wishes to thank the Examiner for the detailed remarks. Claims 1-10 have been cancelled. New claims 11-27 are presented. Accordingly claims 11-27 are pending.

The enclosed substitute specification is being filed pursuant to 37 CFR §1.125 and MPEP 608.01(q). The enclosed marked-up specification showing additions and deletions is a true copy typed identically to the application as filed. The substitute specification includes no new matter and includes the same changes as are indicated in the enclosed marked-up specification.

Claims 1-10 were rejected under 35 U.S.C. §102(b) as being anticipated by *Walkowe* (5553514). Applicant respectfully traverses this rejection. Notably, the Examiner identifies *Walkowe* as having at least two independently rotatable masses (21, 25), however, *Walkowe* recites that element 21 is a flywheel mounted on bearings 22 so that it is capable of rotating with respect to shaft on the same axis of rotation while element 25 is an eddy current brake attached to rotate *with* shaft 12 adjacent to flywheel 21. That is, the eddy current brake 25 is not independently rotatable as recited in Applicant's claims. Although Applicant has cancelled originally-filed claims 1-10, the new claims recite a multiple of independently rotatable masses such that claims 11-27 also are properly allowable over the cited references for at least this reason alone. Accordingly, the claims are properly allowable.

Applicant respectfully submits that this case is in condition for allowance. If the Examiner believes that a teleconference will facilitate moving this case forward to being issued, Applicant's representative can be contacted at the number indicated below.

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ACTIVE FORCE GENERATION SYSTEM FOR MINIMIZING VIBRATION IN A ROTATING SYSTEM

Technical Field

5 This invention relates to vibration isolators, and more particularly, to ~~a new and useful~~ an isolation system for minimizing in-plane vibrations produced in a rotating system of a rotary-wing aircraft, and still more particularly, to an isolation system that minimizes system weight, aerodynamic drag, and complexity while concomitantly providing active control and adjustment during operation for optimal efficacy across a wide spectrum of operating speeds.

Background of the Invention

10 Vibration isolation or absorption is oftentimes desirable for nulling or canceling vibrations associated with a rotating system. Such vibrations, when left unattenuated or unabated, may lead to structural fatigue and premature failure of system components. Furthermore, inasmuch as such vibrations may be transmitted through adjacent support structure to, for example, an aircraft avionics bay, ~~or to areas occupied by passengers, or~~
15 other components and cabin area remote from the source of the vibration, which may also be subject to these same potentially damaging or disturbing vibrations (albeit perhaps lower in amplitude due to energy absorption by the interconnecting structure). Consequently, it is most desirable to isolate or absorb these vibrations at or near the source of the vibration in the rotating system

20 One application which best exemplifies the need for and advantages derived from vibration isolation/absorption devices is the main torque driving hub of a helicopter rotor system. Typically, the main rotor of a helicopter, which comprises a central torque drive hub member for driving a plurality of lift producing rotor blades, is subject to a variety of aerodynamic and gyroscopic loads. For example, as each rotor blade advances or retreats
25 relative to the freestream airflow, it experiences a sharp rise and fall of in-plane aerodynamic drag. Furthermore, as the tip of each rotor blade advances with each

revolution of the rotor system, the relative velocity ~~seen by~~ of the blade tip approaches supersonic Mach numbers. As such, large variations ~~are caused to occur~~ in the various coefficients which define blade performance (e.g., moment, lift and drag coefficients). Moreover, gyroscopic and Coriolis forces are generated causing the blades to “lead” or “lag” depending upon cyclic control inputs ~~made to~~ the rotor system. All of the above generate substantial in-plane and out-of-plane vibrations which, if not suppressed, isolated or otherwise abated, are transmitted to the cockpit and cabin, typically through the mounting feet of the helicopter main rotor gearbox.

Various vibration isolation systems have been devised to counteract/oppose and minimize these in-plane and out-of-plane vibrations. Mast-mounted vibration isolators suppress or isolate in-plane vibrations at a location proximal to the source of such in-plane vibrations whereas transmission, cabin or cockpit absorbers dampen or absorb out-of-plane vibrations at a location remotely disposed from the source. Inasmuch as the present invention relates to the isolation of in-plane vibrations, only devices designed to counteract/oppose such vibrations will be discussed herein.

~~Vincent et al. U.S. Pat. Nos. 4,145,936 and 4,225,287 disclose~~ Some mast-mounted vibration isolators ~~having~~ have a plurality of resilient arms (i.e., springs) extending in a spaced-apart spiral pattern between a hub attachment fitting and a ring-shaped inertial mass. Several pairs of spiral springs (i.e., four upper and four lower springs) are mounted to and equiangularly arranged with respect to both the hub attachment fitting and the inertial mass so as to produce substantially symmetric spring stiffness in an in-plane direction. The spring-mass system, i.e., spiral springs in combination with the ring-shaped mass, is tuned in the non-rotating system to a frequency equal to $N * \text{rotor RPM}$ (e.g., 4P for a four-bladed rotor) at normal operating speed, so that in the rotating system it will respond to both $N+1$ and $N-1$ frequency vibrations (i.e., 3P and 5P for a four-bladed rotor). N is the number of rotor blades.

While ~~the~~ these spiral spring ~~arrangement~~ arrangements ~~of the '936 and '287 patents~~ produce a relatively small width dimension (i.e., the spiraling of the springs increases the effective spring rate), the height dimension of each vibration isolator is increased to react out-of-plane loads via the upper and lower pairs of spiral springs. This

increased profile dimension increases the profile area, and consequently the profile drag produced by the isolator. ~~Furthermore, the~~ The spiral springs must be manufactured to precise tolerances to obtain the relatively exact spring rates necessary for efficient operation. ~~As such, such that~~ manufacturing costs ~~are~~ may be increased. Furthermore, these vibration isolators are passive devices which are tuned to a predetermined in-plane frequency. That is, the vibration isolators cannot be adjusted in-flight or during operation to isolate in-plane loads which may vary in frequency depending upon the specific operating regime.

~~U.S. Patent 3,540,809, 4,057,363 & 4,083,654 issued to inventors Konigsberg and Paul, et al., disclose yet a~~ Another general configuration of isolator known as a “bifilar”. ~~These are~~ mast-mounted vibration isolators ~~comprise having a~~ hub attachment fitting connected to and driven by the helicopter rotorshaft, a plurality of radial arms projecting outwardly from the fitting and a mass coupled to the end of each arm via a rolling pin arrangement. That is, a pin rolls within a cycloidally shaped bushing thereby permitting edgewise motion of each mass relative to its respective arm. The geometry of the pin arrangement in combination with the centrifugal forces acting on the mass (imposed by rotation of the bifilar) results in an edgewise anti-vibration force at a 4 per revolution frequency which is out-of-phase with the large 4 per revolution (or “4P” as it is commonly referred to as helicopter art) in-plane vibrations of the rotor hub for a 4 bladed helicopter. The frequency of 4P is the frequency as observed in a nonrotating reference system.

More specifically, pairs of opposed masses act in unison to produce forces which counteract forces active on the rotor hub. In Fig.1, a schematic of a pair of bifilar masses, at one instant in time, are depicted to illustrate the physics of the device. Therein, the masses MI, MII are disposed at their extreme edgewise position within each of the respective cycloidal bushings BI, BII. The masses MI, MII produce maximum force vectors $F/2$, which produce a resultant vector F at the center, and coincident with the rotational axis, of the rotating system. ~~Furthermore, the~~ The combined or resultant force vector F is equal and opposite to the maximum vibratory load vector P active on the rotor at the same instant of time. This condition, when the bifilar produces ~~and an~~ equal and

opposite force F that opposes the rotor load P , reflects ideal operation of the bifilar. Excessive bifilar damping or manufacturing imperfections will cause the bifilar output force F to differ from the disturbing force P produced by the rotor either in magnitude or phase best suited to nullify the rotor loads. This condition ~~will~~ may cause unwanted fuselage vibration. It will also be appreciated that for the masses to produce the necessary shear forces to react the in-plane vibratory loads of the rotor system, counteracting bending moments are also produced. These force couples impose large edgewise bending loads in the radial arms, and, consequently, the geometry thereof must produce the necessary stiffness (EI) at the root end of the arms. As such, these increased stiffness requirements require ~~larger, heavier~~ the relatively large and heavy bifilar arms.

While the bifilar system has proven effective and reliable, the weight of the system, nearly 210 lbs, is detrimental to the overall lifting capacity of the helicopter. ~~For example, the weight of the bifilar system displaces a passenger and/or increases fuel consumption.~~ To appreciate the significance of the increased weight, it has been estimated that for each pound of additional weight, direct operating cost of the helicopter may increase ~~increases~~ by approximately \$10,000.

Furthermore, the pin mount for coupling each mass to its respective radial arm routinely and regularly wear, thus requiring frequent removal and replacement of the cyclical bushings. This increases the Direct Maintenance Costs (DMC) for operating the helicopter, which contributes, to the fiscal burdens of the bifilar system and the helicopter.

Therefore, a need exists for an isolation system to reduce vibrations in a rotating system that isolates a wide spectrum of vibratory loads; especially large amplitude loads, minimizes system weight, reduces aerodynamic drag and reduces DMC.

Disclosure of the Invention

~~It is the object of the~~ The present invention ~~to provides~~ a vibration isolation system which is controllable for varying the range of isolation frequencies:

~~It is another object of the present invention to provide such a vibration isolation~~
5 ~~system~~ which absorbs large amplitude vibrations while minimizing system weight.

~~It is another object of the present invention to provide such a~~ The vibration isolation system ~~which employs simple, easily readily~~ manufactured components ~~for reducing fabrication costs.~~

~~It is another object of the present invention to provide such a vibration isolation~~
10 ~~system which~~ is insensitive to damping and manufacturing imperfections.

Brief Description of the Drawings

A more complete understanding of the present invention and the attendant
15 features and advantages thereof may be had by reference to the following detailed description when considered in conjunction with the accompanying drawings wherein:

Fig. 1 is a schematic of a prior art bifilar isolation device for illustrating certain physical characteristics thereof.

Fig. 2 is a side sectional view of a helicopter main rotor, including a main rotor
20 shaft having an isolation system according to the present invention mounted to the upper mast or shaft extension member of the rotor.

Figs. 3a – 3c depict schematic views of various operating conditions of the inventive isolation system.

Fig. 4 is a side sectional view of one embodiment of the isolation device.

Best Modes for Carrying Out the Invention

25 The isolation system of the present invention is described in the context of a helicopter rotor system, such as that employed in an Army BLACK HAWK helicopter produced by Sikorsky Aircraft Corporation. One skilled in the art, however, will

appreciate that the present invention has utility in any rotating system which produces vibratory loads (noise). The invention is especially useful in rotating systems that produce large vibratory loads that vary depending upon different operating regimes or variable operating speeds.

5 Referring to Fig. 2, the vibration ~~isolator~~ isolation system 10 ~~according to the present invention~~ is disposed in combination with a rotary-wing aircraft main rotor system 2 ~~comprising having~~ a main rotor shaft 4 (rotating system member) that is driven about ~~its~~ a rotational axis 6 by a torque driving transmission 8. In the described embodiment, the rotor system 2 ~~also comprises a propeller type~~ includes a hub 12 having
10 four radial arms that mount to and drive each rotor blade 16. The vibration isolation system 10 is mounted to a flanged end 13 of the main rotor shaft 4 ~~by means of~~ through a hub attachment fitting 18. Vibratory forces active on the main rotor system 2 are generated by a variety of factors, although the dominant vibrations originate from aerodynamic and/or gyroscopic forces generated by each rotor blade 16. A four bladed
15 rotor system produces 3P vibratory loads, i.e., in a single revolution, the magnitude of the load vector varies from a minimum to a maximum value three times in the rotating frame of reference. This resolves into 4P vibration in the nonrotating frame of reference due to the addition of the 1P rotor rotational speed. While a variety of factors influence the vibratory spectrum of a rotor system, such 4P vibrations are generally a result of each
20 rotor blade experiencing maximum lift when advancing and minimum lift when retreating.

Referring to Figs. 2 and 4, the vibration ~~isolator~~ isolation system 10 ~~comprises~~ includes two, essentially coplanar, masses M1, M2, a ~~means~~ drive system 30 for driving the masses M1, M2 about the rotational axis 6 of the main rotor shaft 4, a
25 ~~means~~ control system 40 for issuing control signals to the drive ~~means~~ system 30 to control the rotational speed and relative angular position of the masses M1, M2 and a power source 50 for energizing the drive ~~means~~ system 30 and ~~controller~~ amplifier control system 40.

The masses M1, M2 are (i) disposed at a predetermined distance R from the main
30 rotor shaft axis 6; (ii) driven in the same or opposing rotational direction as the main

rotor shaft axis 6; and (iii) driven at a rotational speed at least 3P greater than the rotational speed 1P of the rotor shaft 4. In the preferred ~~one~~ embodiment, the drive ~~means-system 30~~ preferably ~~comprises~~ includes a pair of electric motors 34a, 34b for driving each of the masses M1, M2 ~~by means of~~ through a relatively small diameter, constant cross-section radial arm 36 (shown schematically in Fig. 3a – 3c). Moreover, the electric motors 34a, 34b are independent of each other, e.g., may be driven at different rotational speeds to enable variation of the isolation force magnitude and phase.

As shown in Fig. 4, the control ~~means-system~~ 40 requires a speed sensor 42 for issuing signals 42s indicative of the rotational speed 1P of the rotor shaft 4, and a signal processing and amplifier ~~means~~ 44, responsive to the speed signals 42s, ~~for issuing to~~ issue control signals 44s to the drive ~~means-system~~ 30 indicative of the rotational velocity and relative angular position of each of the masses M1, M2. While the speed sensor 42 may be a dedicated unit for sensing rotor speed, the same information may be obtained from a transmission alternator or generator 50 which turns at a predefined speed multiple relative to the rotor speed. The alternator or generator 50 supplies power to the controller-amplifier 44 through the slipring 54. Hence, the ~~controller-control system~~ 40 may use voltage phase information from such devices to issue the appropriate control signals to the drive ~~means-system~~ 30.

While the isolation system 10 may employ a ~~simple-controller-amplifier-control~~ system 40 having a predefined schedule or model of the vibrations, e.g., at prescribed rotor speeds, ~~the preferred-another~~ embodiment may also employs-employ a vibration sensing device or system. As such, the control ~~means-system~~ 40 ~~further-comprises~~ includes one or more vibration feedback sensors 51 for issuing vibration signals 51s indicative of the vibrations (e.g., amplitude, frequency and phase) of the helicopter rotor hub 12. The control ~~means-system~~ 40, therefore, samples vibration levels at predefined intervals or rates to identify a trend - positive (lower vibration levels) or negative (larger vibration levels). Accordingly, as vibration levels change, the ~~controller-control system~~ 40 issues modified signals 44s to the drive ~~means-system~~ 30 until an optimum combination of rotational speed, force magnitude and phase are achieved.

The isolation system 10 may be powered by any of a variety of known methods, especially methods which may require transmission from a stationary to a rotating reference-system field. In the described embodiment shown in Fig. 4 the drive ~~system 30~~ and control-means ~~system 40, 30 and 40~~, respectively, are powered by a 15 kVa generator 50 which provides a 115 volt potential at 400 Hz and with 3 phases (~~standard-typical~~ AC power for helicopters). Power is transferred from the stationary system to the rotating system via a conventional cylindrical slip ring 54. Only a small amount of additional weight is required inasmuch as the slip ring 54 is pre-existing and used for powering other systems e.g., rotor blade de-ice system. This slip ring may also be used to communicate the control signals 42s to the drive ~~means-system 30~~ when the ~~controller~~ control system 40 is mounted in the fuselage rather than on the rotor system 2.

In operation, the masses M1, M2 (shown in Figures 3a-3c) are driven by the drive ~~means-system 30~~ at a rotational speed greater than the rotational speed of the rotating system and appropriately positioned to yield a load vector P10 which is equal and opposite to the load vector P2 produced by the rotor system 2. This counteracting load vector P10 can be viewed as a vector which attempts to cancel or null the displacement of the rotor shaft 4. In the described embodiment, the masses turn at a rotational speed.

Inasmuch as the drive ~~means-system 30~~ is mounted directly to the rotating shaft 4 of the rotor system 2, the drive ~~means-system 30~~ need only drive the masses M1, M2 three additional revolution per cycle (for each revolution of the rotor system) to achieve the desired 4P frequency. That is, since the masses M1, M2 are, in a rotating reference system, driven at one revolution per cycle by the rotor system 2 itself, the drive means 30 need only augment the rotational speed by the difference ($4P - 1P$) to achieve the necessary 4P in the stationary reference system.

Figs. 3a – 3c depict various operating positions of the masses M1, M2 to emphasize the function and versatility of the isolation system ~~49~~10.

In Fig. 3a, the masses M1, M2 are essentially coincident and act in unison to produce a maximum force vector $P10_{MAX}$.

In Fig. 3b, the masses M1, M2 define a right angle (90 degrees) therebetween thereby producing a force vector $P10_{MAX}/(\text{sqrt}(2))$ that is a fraction of the magnitude of the maximum force vector.

In Fig. 3c, the masses M1, M2 define a straight angle (180 degrees) and are essentially opposing to cancel the vectors produced by each of the masses M1, M2 independently or individually.

In Figure 4 the controller 40 issues signals to the drive ~~means-system~~ 30 to (a) drive the masses M1, M2 at a rotational speed greater than that of the rotating system and (b) produce a counteracting load of the correct magnitude and phase to efficiently isolate vibrations.

The ability to independently vary the relative angular position of the masses M1, M2 is especially valuable in applications wherein the magnitude of the vibratory load active in/on the rotating system varies as a function of operating regime or operating speed. In a ~~rotorcraft~~ rotary-wing aircraft, for example, it is common to require the highest levels of vibration isolation in high speed forward flight i.e., where the rotor blades are experiencing the largest differential in aerodynamic loading from advancing to retreating sides of the rotor system. Consequently, it may be expected that the drive ~~means-system~~ 30 produce the maximum load vector $P10_{MAX}$ such as illustrated in Fig. 3a. In yet another example, it is anticipated that the lowest levels of vibration isolation would occur in a loiter or hovering operating mode, where the rotor blades are exposed to the ~~same-generally equivalent~~ aerodynamic and gyroscopic affects. Consequently, it may be expected that the drive means 30 produce no or a minimum load vector $P10_{MIN}$ such as illustrated in Fig. 3c.

Thus far, the discussion herein has concentrated on the rotational speed and angular position of the masses M1, M2 to produce vibration isolation. While this feature of the invention is a primary aspect of the invention, the configuration of the inventive isolation system 10 produces counteracting load vectors P10 which act though the rotational axis of the rotor shaft 4. That is, the line of action of the load vector P10, whether the masses M1, M2 are coincident or opposing, intersects the rotational axis and produces pure radial loads. As such, the radial arms of the isolation system 10 are

principally loaded in tension rather than a combination of tensile and bending moment loads ~~(as seen in the prior art)~~. A consequence of this loading condition is a reduction in system weight inasmuch as the radial arms 36 need not produce high edgewise strength to react bending moment loads.

5 Furthermore, tensile loading in the radial arms 36 enables the use of a constant-cross-section structure to react the centrifugal loads produced by each of the masses M1, M2. Moreover, directional strength materials (non-isotropic) may be employed such as unidirectional fiber reinforced composites. As a result, the isolation device may be produced using relatively low cost manufacturing techniques and materials. For
10 example, cylindrical raw material stock, cut to the proper length, may be ~~purchased and~~ employed without secondary processing. Also, the use of unidirectional composites enables yet further weight reduction

 Although the invention has been shown and described herein with respect to a certain detailed embodiment of a mast-mounted helicopter isolator, it will be understood
15 by those skilled in the art that a variety of modifications and variations are possible in light of the above teachings. It is therefore to be understood that, within the scope of the appended claims, the present invention may be practiced otherwise than as specifically described hereinabove.